**SPECIFICATIONS** 

# **GROUP 9**

# BRAKES

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# BRAKE SPECIFICATIONS, DESCRIPTION, OPERATION **CONTENTS OF SECTION 9-A**

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# **SERVICE BULLETIN REFERENCE**

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### 9-1 BRAKE SPECIFICATIONS

## a. Tightening Specifications

Use a reliable torque wrench to tighten the parts listed, to insure proper tightness without

straining or distorting parts. These specifications are for clean and lightly lubricated threads only; dry or dirty threads produce increased friction which prevents accurate measurement of tightness.

Part	Name	Size	Ft. Lbs.
Nut	Backing Plate to Steering Knuckle Bolt		45-50
Nut	Backing Plate to Axle Housing Bolt	<b>7∕6-20</b>	35-40
Bolt	Wheel Cylinder to Backing Plate	5∕16-18	10-15
Nut	Brake Anchor Pin	<b>5</b> %-18	65-70
Bolt	Brake Drum to Axle Shaft	5/6-18	10-15
Stud	Brake Drum to Axle Shaft (Pilot)	5/6-18	10-15
Nut	Master Cylinder to Frame Bolt		35-40
Nut	Master Cylinder Filler Cap	1 1/4-18	15- <b>20</b>
Nut	Master Cylinder Push Rod Lock	3/8-24	20-25
Bolt	Pedal Shaft Bracket to Frame	<b>3</b> %-16	20-25
Screw	Pedal Shield to Frame Member		10-15
Nut	Brake Pipe Distributor Fitting	5/6-24	10-15
Nut	Brake Hose Bracket to Frame	5/6-24	10-15
Screw	Brake Pipe Tee Fitting to Strut Rod	<sup>5</sup> √6-18	10-15
Bolt	Brake Lever to Dash Brace	5∕ <sub>16</sub> -18	10-15

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#### b. General Specifications

ltems .	Series 40-50-60	Series 70	
Operating Mechanism, Service Brakes	← Hydraulic →		
Parking Brakes	Lever and C	ables	
Operation of Service Brakes Independent of Parking Brakes	Yes	Yes	
Wheels Braked, Service	Front and l	Rear	
Parking	←Rear On		
Approx. % of Total Braking Power on—Front Wheel Brakes	53	53	
Rear Wheel Brakes	47	47	
Static Pressure in Hydraulic System when Brakes are Released	<b>←</b> 8 to 16 lb		
Number of Brake Shoes at Each Wheel	2	2	
Brake Shoe Type	Self Energizing	-Servo	
Brake Shoe Lining Type	1 pc. Molded-F		
Front Shoe Lining Width x Thickness	2.25" x .187"	2.50" x .25"	
Rear Shoe Lining Width x Thickness, Series 40 only	1.75" x .187"		
Rear Shoe Lining Width x Thickness, All other Series	2.25" x .187"	2.25" x .187"	
Brake Shoe and Drum Nominal Diameter	← C.I. Rim Fused to	12'	
Master Cylinder Size	C.I. Rim Fused to	Steel Disk——	
Wheel Cylinder Size, Front	1 1/8"	1 1/8"	
Rear	1.78	1 78	
Approved Hydraulic Brake Fluid	←G.M. or Delco Super	No. 11	
Fluid Level, Below Top of Filler Opening	1/2" to 1		
Brake Pedal to Toeboard Clearance, with Push Rod Against Stop	1,	1'	
Shoe Adjusting Screw Setting, from Point where Wheels can just be turned	_	-	
by hand	<b>←</b> —Back Off 15-17	Notches	
Brake Drum Inside Diameter	← 11.997" to 12	2.003*	
Brake Drum Rebore, Max. Allowable	<b>←</b> 060' over	ize	
Max. Allowable Runout Inside Drum, New or Rebored	.005*	.005"	
Max. Allowable Out-of-Round, Before Rebore	.010*	.010*	
Max. Allowable Out-of-Balance of Drum	<b>←</b> 6 in. oz		
Max. Allowable Space Between Lining and Shoe Rim after Riveting	.005"	. <b>00</b> 5″	
Max. Allowable Space Between Lining and Drum when Shoe is Tested in		****	
Drum	.004"	.004"	

### 9-2 DESCRIPTION OF BRAKE **MECHANISM**

The brake mechanism includes a brake drum and a brake assembly at each wheel, and two separate and independent control systems for applying the brakes—(1) Parking brake control system (2) Service brake control system.

#### a. Wheel Brake Assemblies

Each brake drum consists of a cast iron rim fused to a pressed steel disk. The cast iron rim provides an ideal braking surface and increased brake lining life. An external web around the circumference prevents distortion and aids in dissipation of heat.

The brake assembly at each wheel uses a primary (front) and a secondary (rear) brake shoe of welded steel construction, with one-piece molded lining attached by tubular rivets. The primary shoe lining is shorter than the secondary shoe lining and is of different composition; therefore the two shoes are not interchangeable. See figure 9-1.

Each brake shoe is held against the backing plate by a hold-down spring, pin, and cup which allow free movement of the shoe. The notched upper end of each shoe is held against the single anchor pin by a heavy coil spring. An adjusting

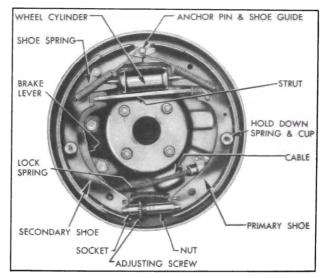


Figure 9-1—Rear Wheel Brake Assembly—Right

screw and lock spring connects the lower ends of both shoes together and provides adjustment for clearance with the brake drum.

A hydraulic wheel cydinder mounted on the backing plate between the upper ends of the brake shoes forces the shoes against the brake drum when the service brakes are applied. On rear wheels only, a lever mounted on each secondary shoe and connected to the primary shoe by a strut is used for applying the shoes when used as parking brakes. See figure 9-1.

necting adjusting screw.

When the brake shoes contact the rotating drum, in either direction of car travel, they move with the drum until the rearward shoe is stopped by the anchor pin and the forward shoe is stopped by the rearward shoe through the connecting adjusting screw. Frictional force between drum and shoe lining tries to rotate each shoe outward around its anchor point but the drum itself prevents this rotation; consequently the shoes are forced more strongly against the drum than the applying force is pushing them. See figure 9-2. It is also evident that the force applied by the drum to the forward shoe is imparted to the rearward shoe through the con-

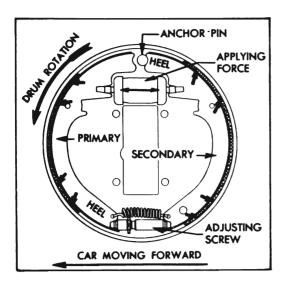


Figure 9-2—Brake Shoe Action

Utilization of the frictional force to increase the pressure of shoes against the drum is called self-energizing action. Utilization of force in one shoe to apply the opposite shoe is called servo action. The self-energizing servo action of Buick brakes provides powerful braking action with relatively light pedal pressure.

#### b. Parking Brake Control System

The parking brake control system uses a foot operated brake lever, conduit enclosed cables, brake shoe levers and struts to apply the rear wheel brakes only. The brake lever cable connects to a sheave (equalizer) located at center of the rear brake cable, with an adjusting nut to take up cable slack. Each end of the rear brake cable is attached to the free lower end of a brake shoe lever pivoted on each secondary (rear) brake shoe. A strut is mounted between each brake shoe lever and the primary (front) brake shoe. See figure 9-3.

When the brake lever is pushed forward the cables apply an equal pull to each brake shoe lever, and the levers and struts force all rear brake shoes into firm contact with the brake drums. A spring loaded latch automatically locks the brake lever to keep the parking brakes applied. The brake lever is released by pulling the lever release knob. A warning signal, which is standard on Series 70 and optional on Series 40-50-60, will show a red light on instrument panel if the car is operated with the parking brakes applied.

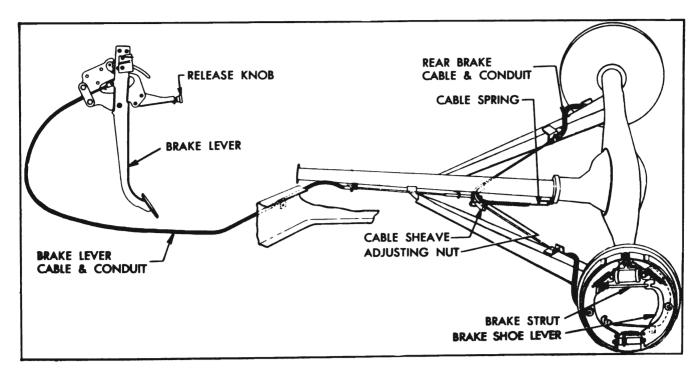


Figure 9-3—Parking Brake Control System

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## c. Service Brake Control System— Foot Powered

NOTE: See paragraph 9-15 for power brakes. The regular foot powered service brake control system is a pedal operated hydraulic system which applies the brakes at all four wheels with equalized pressure.

The hydraulic system consists of one master cylinder connected by pipes and flexible hoses to a wheel cylinder mounted between the brake shoes at each wheel. The master cylinder, pipes, hoses, and four wheel cylinders are filled with a special fluid. See figure 9-4.

Each wheel cylinder contains two pistons and two rubber cups which are held in contact with the pistons by a central coil spring to provide a fluid-tight seal. The inlet port for brake fluid is located between the pistons so that when fluid pressure is applied both pistons move outward towards the ends of wheel cylinders. The pistons impart movement to the brake shoes by means of connecting links which seat in pistons and bear against webs of shoes. Rubber boots enclose both ends of cylinder to exclude foreign matter. See figure 9-5. A valve for bleeding the brake pipes and wheel cylinder is located above the inlet port.

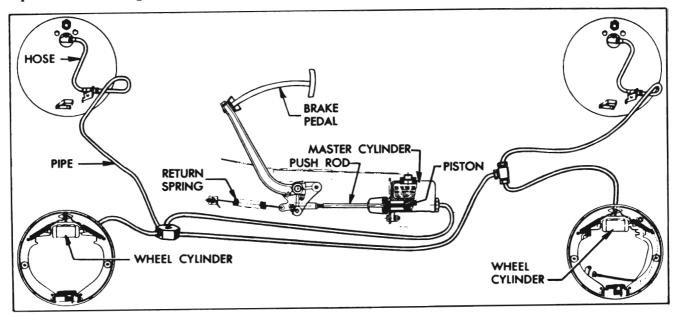


Figure 9-4—Service Brake Control System

The master cylinder contains a vented reservoir for reserve fluid and a cylinder or pressure chamber in which force applied to the brake pedal is transmitted to the fluid which actuates the brake shoes. A breather port and a compensating port permits passage of fluid between the reservoir and pressure chamber under certain operating conditions. The outlet end of the pressure chamber is closed by a head nut and gasket. A coil spring holds a check valve against the head nut and also holds a rubber primary cup against the inner end of the piston. This cup and a rubber secondary cup on the outer end of the piston prevent escape of fluid past the piston. The piston is retained in the cylinder by a stop plate and lock ring. The push rod which actuates the piston extends through the stop plate, and a rubber boot is installed over this end of the cylinder to exclude foreign matter. See figure 9-5, view A.

# 9-3 OPERATION OF HYDRAULIC SERVICE BRAKES

NOTE: See paragraph 9-15 for power brakes. When the brakes are fully released, the master cylinder piston is held against the stop plate and the primary cup is held just clear of the compensating port by the master cylinder spring, which also holds the check valve against its seat on the head nut. The pressure chamber is filled with fluid at atmospheric pressure due to the open compensating port and reservoir vent. All pipes and wheel cylinders are filled with fluid under a "static" pressure of 8-16 pounds, which holds the lips of the wheel cylinder cups in firm contact with cylinder walls to prevent loss of fluid or entrance of air. See figure 9-15, view A.

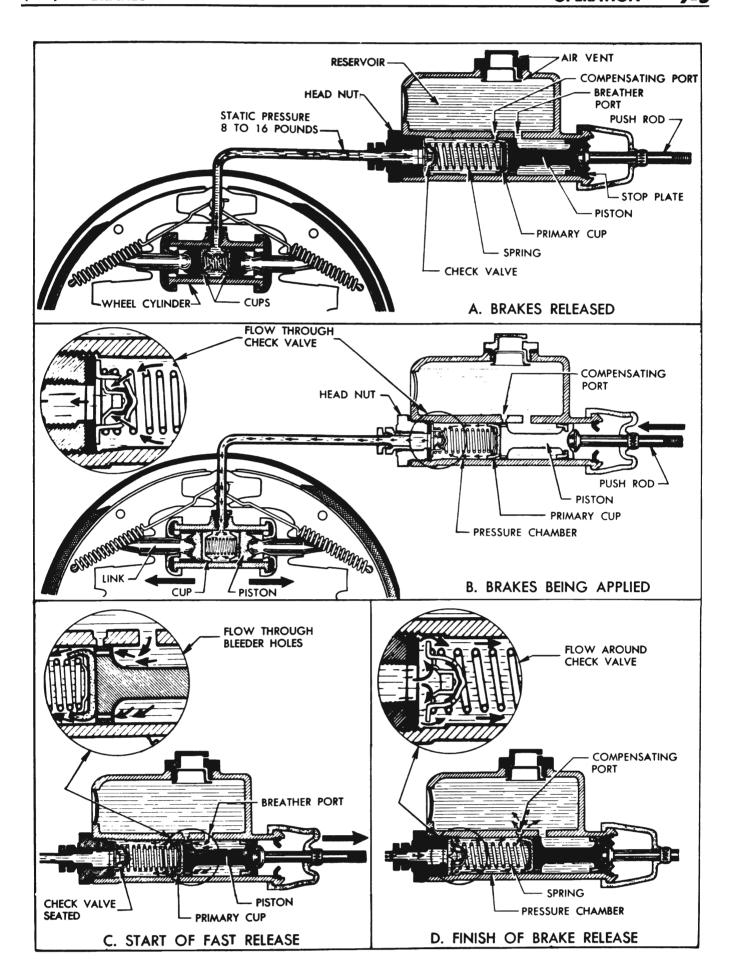


Figure 9-5—Operation of Brake Hydraulic System

When the brake pedal is depressed to apply the brakes, the push rod forces the master cylinder piston and primary cup toward the head nut. As this movement starts, the lip of the primary cup covers the compensating port to prevent escape of fluid into the reservoir. Continued movement of the piston builds pressure in the pressure chamber and fluid is then forced through holes in the check valve and out into the pipes leading to all wheel cylinders. Fluid forced into the wheel cylinders between the pistons and cups causes the pistons and connecting links to move outward and force the brake shoes into contact with the drums. See figure 9-5, view B.

Movement of all brake shoes into contact with drums is accomplished with very light pedal pressure. Since pressure is equal in all parts of the hydraulic system, effective braking pressure cannot be applied to any one drum until all of the shoes are in contact with their respective drums; therefore the system is self-equalizing. After all shoes are contacting the drums, further force on brake pedal builds up additional pressure in the hydraulic system, thereby increasing the pressure of shoes against drums.

On rapid stops some car weight is transferred from the rear to the front wheels, consequently greater braking power is required at front wheels in order to equalize the braking effect at front and rear wheels. Greater pressure is applied to front brake shoes by using larger wheel cylinders, so that distribution of braking power is approximately 56% at front wheels and 44% at rear wheels.

When the brake pedal is released, the pedal return spring pulls the pedal back until the push

rod contacts the stop plate in master cylinder. The master cylinder spring forces the piston and primary cup to follow the push rod and also presses the check val firmly against its seat.

At start of a fast release the piston moves faster than fluid can follow it in returning from the pipes and wheel cylinders, therefore, a partial vacuum is momentarily created in the pressure chamber. Fluid supplied through the breather port is then drawn through the bleeder holes in piston head and past the primary cup to keep the pressure chamber filled. See figure 9-5, view C.

As pressure drops in master cylinder the shoe springs retract all brake shoes and the connecting links push the wheel cylinder pistons inward, forcing fluid back to master cylinder. Pressure of returning fluid causes a rubber disc to close all holes in the check valve and forces the check valve off its seat against the tension of master cylinder spring; fluid then flows around the check valve into the pressure chamber. With the piston bearing against the stop plate and the lip of the primary cup just clear of the compensating port, excess fluid which entered through the bleeder holes, or was created by expansion due to increased temperature, now returns to reservoir through the uncovered compensating port. See figure 9-5, view D.

When pressure in wheel cylinders and pipes becomes slightly less than the tension of master cylinder spring, the check valve returns to its seat on head nut to hold 8 to 16 pounds of "static" pressure in the pipes and cylinders.